

Chap. 3
ATS 541 HW ~~111~~ 3

Solutions

~~100~~ ~~65~~ pts. total

	1.	20	(grads only)
	2.	12	
	3.	10	
	4.	10	
	5.	5	
	6.	10	
	7	8	10
5.4	4.2	10	20
5.7	4.5	10	10
5.10	4.7	15	10

~~97~~ 100

Total: 97 for undergrads
117 for grads

HW2 ch 3 (Krupp's Notes, Ch. 4 Tseris)

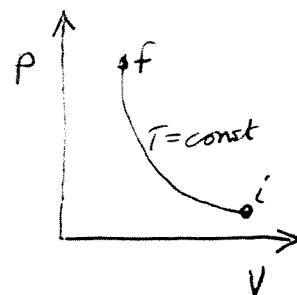
①

1. (a) Isothermal reversible compression

pts. 20 $W = \int_{V_i}^{V_f} p dV$

$$pV = nRT$$

$$V = \frac{nRT}{p}$$



$$dV = -nRT \frac{dp}{p^2} \quad \left[+ \frac{nR}{p} dT \right]$$

$$W = \int_{p_i}^{p_f} -nRT \frac{dp}{p}$$

$$W = -nRT \ln \frac{p_f}{p_i} \quad \text{or} \quad W = nRT \ln \left(\frac{V_f}{V_i} \right)$$

$$\Delta u = \int du = \int_{T_i}^{T_f} C_v dT = 0 \quad \text{since } T = \text{const.}$$

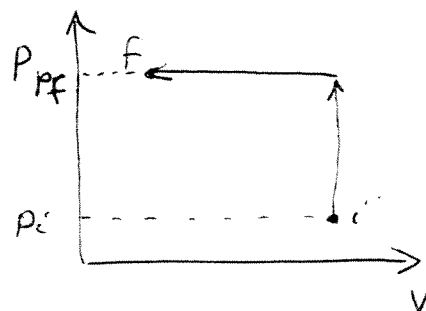
b) sudden compression to p_f (from p_i)

An example like this was given in the notes.

$$W = \int_{V_i}^{V_f} p dV = \underbrace{p_{\text{ext}} (V_f - V_i)}_{\text{same}} = p_f (V_f - V_i)$$

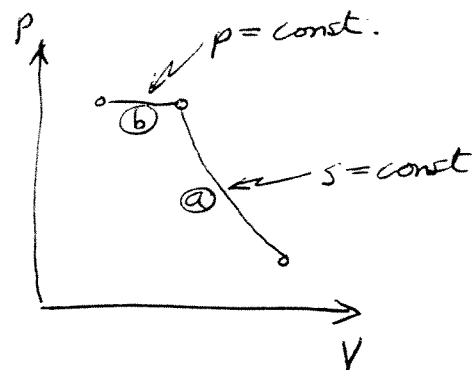
$$W = p_f (V_f - V_i)$$

Again, $\Delta u = 0$ since $T_f = T_i$



c) adiabatic reversible compression followed by a reversible isobaric cooling. (2)

Need to consider 2 processes
 (a) and (b).



Process (a) is adiabatic

$$dq = 0 \rightarrow dw_a = -du$$

Process (b) is const. pressure, so $dw_b = p dV$

$$\therefore W = W_a + W_b$$

$$= -(u' - u_i) + p_f (V_f - V')$$

$$= -nc_v(T' - T_i) + p_f(V_f - V')$$

$$= -nc_v(T' - T_i) + nR(T_f - T') \leftarrow pV = nRT$$

$$= (nc_v T + nRT) - (nc_v T' + nRT')$$

$$= nc_p(T - T')$$

$$[C_p = C_v + R]$$

$$= -nc_p T \left(\frac{T'}{T} - 1 \right)$$

$$\frac{T'}{T_i} = \left(\frac{p'}{p_i} \right)^{R/C_p} \quad (p_f = p')$$

$$W = -nC_p T \left[\left(\frac{p_f}{p_i} \right)^{R/C_p} - 1 \right] \checkmark$$

$$\Delta u = 0 \quad \text{Since } T_f = T_i \quad \checkmark$$

d.)

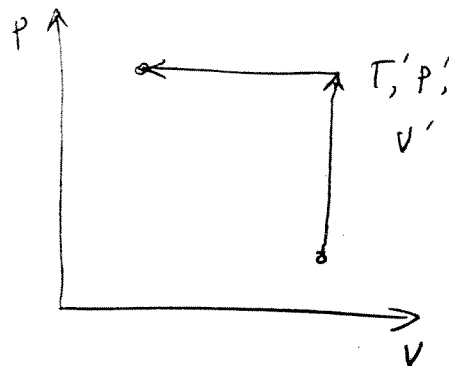
① rev. increase of T @ const V [Followed by]② rev. decrease of P @ const P

③

Process ①:

$$W_1 = 0 \quad \text{since } V' = V_i$$

$$\Delta U = n c_v (T' - T_i)$$



Process ②:

$$W_2 = \int_{V_i}^{V_f} p dV = \overset{\text{sign correction}}{-P_f (V_i - V_f)} = -P_f \left[\frac{nRT_i}{P_i} - \frac{nRT_f}{P_f} \right]$$

$$T_i = T_f = T$$

$$W_2 = nRT \left(\frac{P_f}{P_i} - 1 \right)$$

$$\Delta U = n c_v (T' - T_f)$$

$$W = W_1 + W_2 = nRT \left(\frac{P_f}{P_i} - 1 \right)$$

$$\Delta U = 0$$

Grading:

1 pts: $\Delta U = 0$
in all cases4 pts. ea: W in
each part.

2.

12 pts

$$p_1 = 1000 \text{ mb}$$

$$T_1 = 10^\circ\text{C}$$

dry air

$$p_2 = 700 \text{ mb}$$

This is an adiabatic process, so $dq = 0$.

$$\begin{aligned} \text{a) } \alpha_1 &= \frac{R_d T_1}{p_1} = \frac{(287 \text{ J kg}^{-1} \text{ K}^{-1})(283.15 \text{ K})}{10^5 \text{ Pa}} \\ &= \boxed{0.81 \text{ m}^3 \text{ kg}^{-1}} \end{aligned}$$

$$\begin{aligned} \text{b) } T_2 &= T_1 \left(\frac{p_2}{p_1} \right)^{R_d/c_p} = (283.15 \text{ K}) \left(\frac{700}{1000} \right)^{-2.86} \\ &= \boxed{255.6 \text{ K}} \end{aligned}$$

$$\alpha_2 = \frac{R_d T_2}{p_2} = \frac{(287)(255.6)}{7 \times 10^4 \text{ Pa}} = \boxed{1.05 \text{ m}^3 \text{ kg}^{-1}}$$

$$\begin{aligned} \text{c) } \Delta u &= c_v \Delta T \\ &= (717 \text{ J kg}^{-1} \text{ K}^{-1})(255.6 - 283.1) \text{ K} \\ &= \boxed{-1.97 \times 10^4 \text{ J kg}^{-1}} \end{aligned}$$

$$\begin{aligned} \Delta h &= c_p \Delta T \\ &= (1005 \text{ J kg}^{-1} \text{ K}^{-1})(255.6 - 283.1) \text{ K} \\ &= \boxed{-2.76 \times 10^4 \text{ J kg}^{-1}} \end{aligned}$$

$$\text{d) } dw = p d\alpha \text{ (adiabatic)} \rightarrow dw = -du$$

$$\Delta W = -\Delta u = -1.97 \times 10^4 \text{ J kg}^{-1} \text{ from part (c)}$$

Find mass in 1 km^3 @ $\frac{1000}{700} \text{ mb}$

$$m = \frac{V}{\alpha} = \frac{1 \text{ km}^3}{0.81 \text{ m}^3 \text{ kg}^{-1}} = \frac{(10^3 \text{ m})^3 / \text{km}^3}{0.81 \text{ m}^3 \text{ kg}^{-1}} = 1.23 \times 10^9 \text{ kg}$$

$$\therefore W = m \Delta u = (1.23 \times 10^9 \text{ kg})(-1.97 \times 10^4 \text{ J kg}^{-1}) = \boxed{-2.43 \times 10^{13} \text{ J}}$$

[3.] (10 pts)

~~11/8~~ $h = u + p\alpha$

a) Prove dh is exact

Assuming ideal gas, $p\alpha = RT$

$$h = u + RT$$

$$dh = du + RdT$$

$$= c_v dT + RdT$$

$$= (c_v + R) dT \quad (= f(T))$$

$$dh = c_p dT$$

Exact, as is $du = c_v dT$.

b) $p_1 = 70 \text{ kPa}$ $p_2 = 100 \text{ kPa}$

$$T_1 = 10^\circ\text{C} = 283.15 \text{ K}$$

adiabatic compression $\Rightarrow T_2 = T_1 \left(\frac{p_2}{p_1}\right)^{\kappa}$

$$= 283.15 \left(\frac{100}{70}\right)^{\kappa}$$

$$= 313.56 \text{ K}$$

$$\int_1^2 dh = \Delta h = \int_{T_1}^{T_2} c_p dT = c_p \Delta T$$

$$= (1005.7 \text{ J kg}^{-1} \text{ K}^{-1}) (313.56 - 283.15) \text{ K}$$

$$= 3.058 \times 10^4 \text{ J kg}^{-1}$$

4.
~~4.~~

Cold case

O_e simple 251.87

O_e Bolton 257.90

Warm case

O_e simple 362.1

O_e Bolton 368.4

5. Specific heat definition

5 pts $c = \frac{dq}{dT}$

For adiabatic, $dq = 0 \Rightarrow c = 0$

For isothermal $dq \rightarrow \infty \rightarrow$ undefined

$$\lim_{\Delta T \rightarrow 0} \frac{dq}{dT} = \infty$$

6. Cruising altitude ~ 10 km (35,000 ft)

a) Assume $T_{\text{sea}} \approx 25^\circ\text{C}$

10 pts. $\left(\frac{\partial T}{\partial z}\right) \approx 6.5^\circ\text{C}/\text{km}$

$$\begin{aligned}\Rightarrow T_{10\text{km}} &= T_{\text{sea}} - \frac{\partial T}{\partial z} \cdot 10\text{ km} \\ &= 25^\circ\text{C} - \cancel{10}\text{ km} \cdot 6.5^\circ\text{C}/\text{km} \\ &= 25 - 65 = -40^\circ\text{C} = 233\text{ K}\end{aligned}$$

b) Assume $P_{\text{outside}} = 25\text{ kPa}$

If this air is brought inside, via adiabatic compression, then $T_{\text{in}} = T_{\text{out}} \left(\frac{P_{\text{in}}}{P_{\text{out}}}\right)^{\kappa}$

$$= 233 \left(\frac{85}{25}\right)^{-2.86}$$

$$= 330.6\text{ K} = 57.6^\circ\text{C}$$

\therefore Air must be cooled.

7.

a) Heated air expands

~~8~~ pts.

From the eq. of state, $p\alpha = RT$, we see that, for fixed p , $\alpha \propto T$, so α increases as T increases.

10 pts

b) When air expands, it cools.

In this case, p decreases in an adiabatic ~~expansion~~ expansion:

$$dq = 0 = c_v dT + p d\alpha$$

$$c_v dT = -p d\alpha = -dw$$

For $d\alpha > 0$ (expanding air), $dT < 0$

Hence the air cools.

Tsonis 4.2: A sample of 100 g of dry air has initial T of 270 K and $p = 900$ mb. During an isobaric process heat is added and the volume expands by 20% of the initial volume. Estimate (a) final temperature, (b) heat added, and (c) work done against the environment.

4.2 $m = 100$ g $p = \text{const}$ dry air
 $T = 270$ K $V_f = 1.2 V_i$
 $p = 900$ mb $\alpha_f = 1.2 \alpha_i$

$dg = c_v dT + p d\alpha$
 $dQ = m c_v dT + p dV = m c_p dT - m \alpha dp$

$m(c_p - c_v) dT = p dV$
 $m R dT = p dV$

$m R \int_{T_i}^{T_f} dT = \underbrace{p \int_{V_i}^{V_f} dV}_{\text{work}}$

$pV = mRT$
 $p dV = mR dT$

$\Delta T = (T_f - T_i) = \frac{W}{mR}$
 $= \frac{1550 \text{ J}}{(0.1 \text{ kg})(287 \text{ J kg}^{-1} \text{ K}^{-1})}$
 $= 54.0 \text{ K}$

$V_i = \frac{mRT_i}{p_i}$
 $= \frac{(0.1 \text{ kg})(287 \text{ J kg}^{-1} \text{ K}^{-1})(270 \text{ K})}{90,000 \text{ Pa}}$

$V_i = 0.0861 \text{ m}^3$

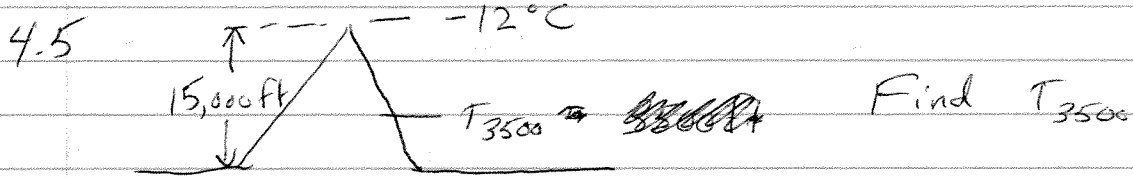
$V_f = 1.2(0.0861 \text{ m}^3) = 0.1033 \text{ m}^3$

(a) $T_f = 270 + 54 = 324 \text{ K}$

(c) $W = (90,000 \text{ Pa})(0.1033 - 0.0861) \text{ m}^3$
 $= 1550 \text{ J}$

(b) $\Delta Q = m c_v \Delta T + W$
 $= (0.1 \text{ kg})(717 \text{ J kg}^{-1} \text{ K}^{-1})(54 \text{ K}) + 1550 \text{ J}$
 $= 3872 \text{ J} + 1550 \text{ J} = 5422 \text{ J}$

Tsonis 4.5: Assume that you are on the top of a mountain at an altitude of 15,000 ft and there are no clouds above or below. If the $T = -12^\circ\text{C}$, what will the temperature be at 3500 ft altitude?



Assume dry air, so $\Gamma_d = -9.8^\circ\text{C}/\text{km}$

Difference in altitude $\Delta h = (15,000 - 3500)\text{ft}$

$$= 11,500\text{ ft}$$
$$= 11,500\text{ ft} * (0.3048 \frac{\text{m}}{\text{ft}})$$
$$= 3.505\text{ km}$$

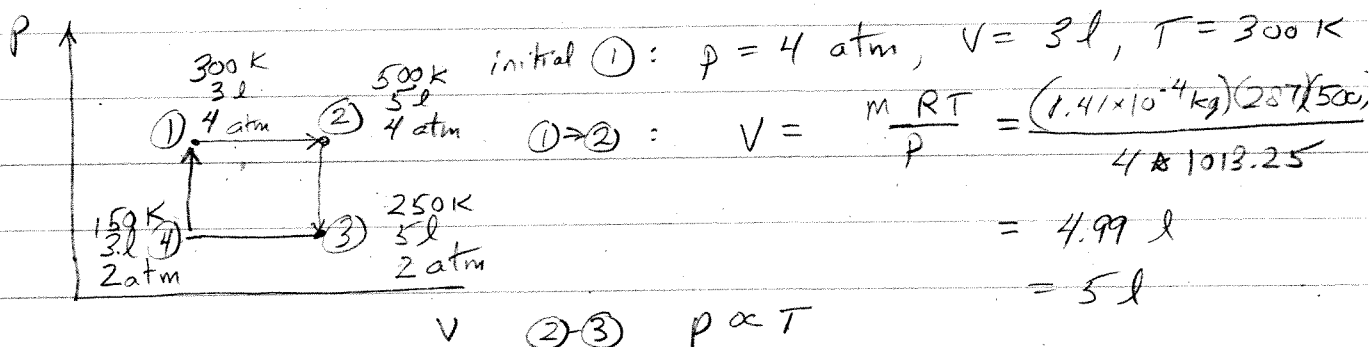
$$T_{3500} = T_{\text{top}} + \Gamma_d (\Delta h)$$
$$= -12^\circ\text{C} + (-9.8^\circ\text{C}/\text{km})(-3.505\text{ km})$$
$$= -12^\circ\text{C} + 34.35^\circ\text{C}$$
$$= \boxed{22.35^\circ\text{C}}$$

Tsonis 4.7: A sample of dry air has a $T = 300 \text{ K}$, a volume = 3 liters, and $p = 4 \text{ atm}$. The air undergoes the following changes: (a) warmed at const p to 500 K, (b) cooled at const V to 250 K, (c) cooled at const p to 150 K, and (d) warmed at const V to 300 K. (1) Describe graphically on a (p,V) diagram each of these changes, (2) calculate the total work done.

- 4.7 dry air
- $T = 300 \text{ K}$
- $V = 3 \text{ l}$
- $p = 4 \text{ atm}$
- a) warmed at $p = \text{const}$ to 500 K
- b) cooled @ $V = \text{const}$ to 250 K
- c) cooled @ $p = \text{const}$ to 150 K
- d) warmed @ $V = \text{const}$ to 300 K

15 pts

$$m = \frac{pV}{RT} = \frac{4(1013.25)(3\text{l})(1\text{m}^3/1000\text{l})}{(287.5 \text{ kg}^{-1}\text{K}^{-1})(300\text{K})} = 1.41 \times 10^{-4} \text{ kg} = .141 \text{ g}$$



$$(dQ = C_v dT + p dV \quad pV = nRT)$$

③-④ $V \propto T$ $V = \left(\frac{150}{250}\right) 5 \text{ l} = 3 \text{ l}$

$$W = p \int_{V_1}^{V_2} dV + p \int_{V_3}^{V_4} dV$$

$$= 4 \text{ atm} (5 - 3 \text{ l}) + 2 \text{ atm} (3 - 5 \text{ l})$$

$$= +8 \text{ l} \cdot \text{atm} - 4 \text{ l} \cdot \text{atm}$$

$$= 4 \text{ l} \cdot \text{atm} = 4 \times 1013.25 \times 100 \times 10^{-3} \text{ m}^3 \text{ l}^{-1}$$

$$= \boxed{405.3 \text{ J}}$$

Petty

5.7 $\theta = 310 \text{ K}$ Find T @ 700, 300, 100 hpa

$$\theta = T \left(\frac{p_0}{p} \right)^{R_d/c_p} \Rightarrow T = \theta \left(\frac{p_0}{p} \right)^{-R_d/c_p}$$

$$a) T = 310 \left(\frac{1000}{700} \right)^{-0.286} = 279.9 \text{ K}$$

$$b) T = 310 \left(\frac{1000}{300} \right)^{-0.286} = 219.7 \text{ K}$$

$$c) T = 310 \left(\frac{1000}{100} \right)^{-0.286} = 160.5 \text{ K}$$

Petty

5.10 $\Delta z = 10 \text{ m}$ $\frac{\Delta T}{\Delta t} = -5 / 10 \text{ hr.}$

$$\rho = 1.25 \text{ kg m}^{-3}$$

Assume $p = \text{const.}$

$$dq = c_p dT - \alpha dp$$

Find m : mass per unit area

$$dQ = m c_p dT$$

$$m = \rho \Delta z$$

$$\dot{Q} = \frac{dQ}{dt} = m c_p \frac{dT}{dt}$$

$$= 12.5 \text{ kg m}^{-2}$$

$$\dot{Q} = (12.5 \text{ kg m}^{-2}) (1005.7) \frac{-5 \text{ K}}{10 \text{ K} \cdot 3600 \text{ s hr}^{-1}}$$

$$= -1.75 = 5 \text{ m}^{-2} \text{ s}^{-1}$$

$$= -1.75 \text{ W m}^{-2}$$

Petty 5.4

20 pts

Compute i) mechanical work and
ii) heat added for the following

a) isothermal compression, $T = 15^\circ\text{C}$, to $\frac{1}{5}$ orig. volume

$$\begin{aligned} \text{i) } W &= \int p d\alpha \quad p\alpha = RT \quad \alpha = \frac{RT}{p} \Rightarrow p = \frac{RT}{\alpha} \\ &= RT \int_1^{0.2} d \ln \alpha = RT \ln(0.2) \\ &= (287)(288)(-1.609) \end{aligned}$$

$$\boxed{W = -1.33 \times 10^5 \text{ J kg}^{-1}}$$

$$\begin{aligned} \text{ii) } dq &= c_v dT + p d\alpha \quad (\text{isothermal}) \Rightarrow dq = dw \\ \Delta q &= w \\ &= \boxed{-1.33 \times 10^5 \text{ J kg}^{-1}} \end{aligned}$$

b) isobaric heating from 0 to 20°C

$$\begin{aligned} dq &= c_p dT - \alpha dp \\ \text{ii) } \Delta q &= c_p \Delta T = 1005.7(20) = \boxed{2.01 \times 10^4 \text{ J kg}^{-1}} \end{aligned}$$

$$\begin{aligned} \text{i) } dq &= c_v dT + dw \\ w &= \int dq - \int c_v dT = \Delta q - c_v \Delta T \\ &= 2.01 \times 10^4 - (717)(20) \\ &= \boxed{5740 \text{ J kg}^{-1}} \end{aligned}$$

c) isochoric heating, 0 to 20°C

$$dq = c_v dT + p d\alpha$$

$$\text{ii) } \Delta q = c_v \Delta T = (717)(20) = 1.43 \times 10^4 \text{ J kg}^{-1}$$

$$\text{i) } w = \int p d\alpha = 0 \quad (d\alpha = 0) \text{ (isochoric)}$$

d) Adiabatic expansion to 5 times initial volume
 $T_i = 20^\circ\text{C}$

$$dq = c_v dT + p d\alpha = 0 \quad (\text{adiabatic})$$

$$c_v dT = -p d\alpha \quad p\alpha = RT \quad p = \frac{RT}{\alpha}$$

$$c_v dT = -RT \frac{d\alpha}{\alpha}$$

$$\int_{T_i}^{T_f} \frac{dT}{T} = -\frac{R}{c_v} \int_{\alpha_i}^{\alpha_f} \frac{d\alpha}{\alpha}$$

$$\ln \frac{T_f}{T_i} = -\frac{R}{c_v} \ln 5$$

$$T_f = T_i \exp\left(-\frac{R}{c_v} \ln 5\right)$$

$$= 293 \exp\left(-\frac{287}{717} \cdot 1.609\right) = 153.9 \text{ K}$$

$$\text{i) } \int dw = w = -c_v \int_{T_i}^{T_f} dT = -c_v (153.9 - 293.15) = 9.9 \times 10^4 \text{ J kg}^{-1}$$

$$\text{ii) } \Delta q = 0 \quad (\text{adiabatic})$$